

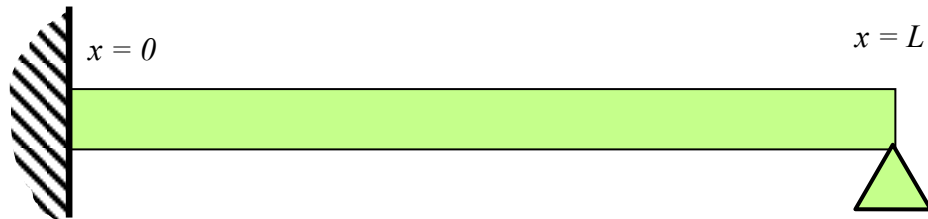
**DYNAMICS (VIBRATION)**

**SHEET 3 : SHAFT WHIRL & BEAM VIBRATIONS**

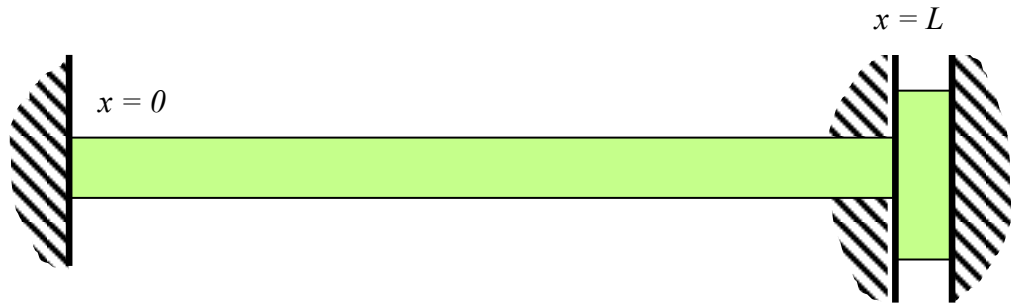
Take  $E = 207 \text{ GN/m}^2$ ,  $\rho = 7800 \text{ kg/m}^3$  for questions 2-5

1. A uniform beam of length,  $L$ , is clamped at one end,  $x = 0$ , and pinned at the other,  $x=L$ , as shown in the figures below. For each beam find:
  - a. Using boundary conditions determine the generalized matrix,  $[Z]\{C\} = \{0\}$ , that could be used to solve for undamped natural frequencies and mode shapes of the beam shown. You do not need to solve for the constants  $\{C\}$ , only show the generalized matrix and terms contained in  $[Z]$ .
  - b. Briefly sketch the resulting displacement of the beam for the first 3 mode shapes that you would expect.

i)



ii) The slider at  $x=L$  allows vertical motion, but not rotation.



iii) There is a point mass at the right hand side of the beam of mass  $m$ . Assume it can rotate and has rotational inertia,  $I_m$ .



iv) For the beam shown in iii) assume the mass on the right hand side of the beam does not rotate (i.e.  $I_m=0$ ).

2. A 25 mm diameter shaft, 1.5 m long, is held by two roller bearings at one end (giving a "clamped" boundary condition) and by a self-aligning ball bearing at the other end (giving a "pinned" boundary condition). Find the critical speeds of the shaft.

2108, 6829, 14250, etc. (rev/min)

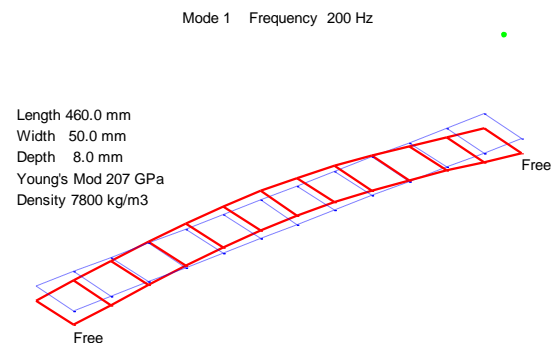
3. Derive the frequency equation for flexural vibration of a uniform beam that is free at both ends.

Using the values of the roots given in the handout, calculate the two lowest natural frequencies for a free-free beam of length 460 mm and a rectangular cross-section 50 mm by 8 mm.

Repeat the calculations using the Matlab script febeam.m, which you can download from the module's WebCT site.

$$\lambda^2 (\cos \lambda L \cosh \lambda L - 1) = 0;$$

200 Hz, 551.8 Hz



4. A beam of length,  $L$ , is clamped at the end  $x = L$ . At  $x = 0$ , a spring of stiffness,  $k$ , is pin-jointed to the beam and provides lateral restraint. Find the frequency equation in determinant form.

$$\begin{vmatrix} 0 & -\lambda^2 & 0 & \lambda^2 \\ -EI \lambda^3 & k & EI \lambda^3 & k \\ \sin \lambda L & \cos \lambda L & \sinh \lambda L & \cosh \lambda L \\ \lambda \cos \lambda L & -\lambda \sin \lambda L & \lambda \cosh \lambda L & \lambda \sinh \lambda L \end{vmatrix} = 0$$

5. Derive the frequency equation for flexural vibration of a uniform beam that is pinned at one end and free at the other. Derive the expression for the mode shape function.

$$\lambda (\tan \lambda L - \tanh \lambda L) = 0; \quad Y_r(x) = \sin \lambda_r x + \frac{\sin \lambda_r L}{\sinh \lambda_r L} \sinh \lambda_r x$$