

Solutions for Exercise Sheet 4: VIBRATION  
Free Vibration

① Checking the damping ratio  $\xi = \frac{c}{2m\omega_n}$

$$\omega_n = \sqrt{\frac{k}{m}} = \sqrt{\frac{500}{2}} = 15.81 \text{ rad/s}$$

$$\xi = \frac{2}{2 \times 2 \times 15.81} = 0.031 \quad (\text{light damping})$$

Damped free vibration:  $x(t) = C \cdot e^{-\xi\omega_n t} \sin(\omega_d t + \phi)$

Initial conditions: i)  $x(0) = 0 = C \cdot \sin \phi \rightarrow \phi = 0$  since  $C \neq 0$ .

ii)  $\dot{x}(0) = v$

$$\dot{x}(t) = -\xi\omega_n C \cdot e^{-\xi\omega_n t} \sin(\omega_d t) + \omega_d C \cdot e^{-\xi\omega_n t} \cos(\omega_d t)$$

$$\dot{x}(0) = v = \omega_d \cdot C \rightarrow C = \frac{v}{\omega_d}$$

Thus,  $x(t) = \frac{v}{\omega_d} \cdot e^{-\xi\omega_n t} \sin(\omega_d t)$  where  $\omega_d = \omega_n \sqrt{1 - \xi^2}$

② The equivalent spring stiffness  $k_e = k + k = 500 \text{ N/m}$ .

$$\omega_n = \sqrt{\frac{k_e}{m}} = \sqrt{\frac{500}{2}} = 15.81 \text{ rad/s}$$

$$\xi (\text{damping ratio}) = \frac{63.24}{2 \times 2 \times 15.81} = 1 \quad (\text{critically damped})$$

$$x(t) = (C + Dt) e^{-\omega_n t}$$

Initial conditions:

i)  $x(0) = C = 0.02$

ii)  $\dot{x}(t) = D \cdot e^{-\omega_n t} + (C + Dt) e^{-\omega_n t} \times (-\omega_n)$

$$\Leftrightarrow \dot{x}(0) = D + C \times (-\omega_n) = 0$$

$$D = C \omega_n = 0.02 \times 15.81 = 0.316.$$

$$\text{So } x(t) = (0.02 + 0.316t) e^{-15.81t} \quad [\text{m}]$$

$$\dot{x}(t) = 0.316 e^{-15.81t} - (0.316 + 4.995t) e^{-15.81t} \quad [\text{m/s}].$$

$$\Leftrightarrow \dot{x}(t) = -4.995t \cdot e^{-15.81t} \quad [\text{m/s}]$$

③ To ensure the system is lightly damped :

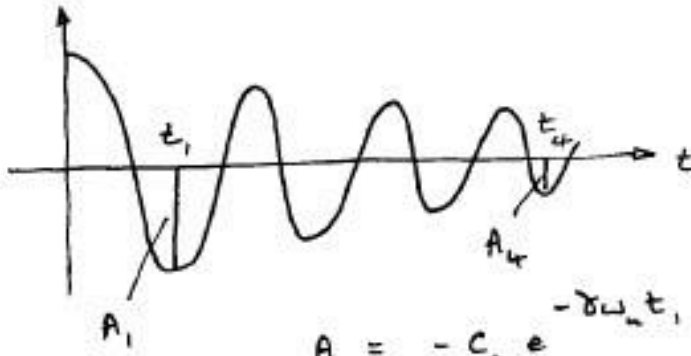
$$\frac{c}{2\sqrt{km}} < 1$$

$$\Leftrightarrow \frac{c}{2\sqrt{km}} < 1 \quad \text{where } c = 63.24 \text{ N s/m}$$

$$\Leftrightarrow c^2 < 4km \quad \longrightarrow km > \frac{c^2}{4} \text{ or } km > 9998 \times 10^2 \left[ \frac{\text{N}^2 \text{kg}}{\text{m}} \right]$$

Since the damper is not changed, either  $k$  or  $m$  can be increased to satisfy the above criterion.

4.



$$A_1 = -C_0 e^{-\delta \omega_n t_1} \quad A_2 = -C_0 e^{-\delta \omega_n t_4}$$

$$\frac{A_1}{A_4} = 4 = e^{\delta \omega_n (t_4 - t_1)} \quad \text{①}$$

But  $t_4 - t_1 = 3T_n$  (where  $T_n = \text{period of free vibration}$ )

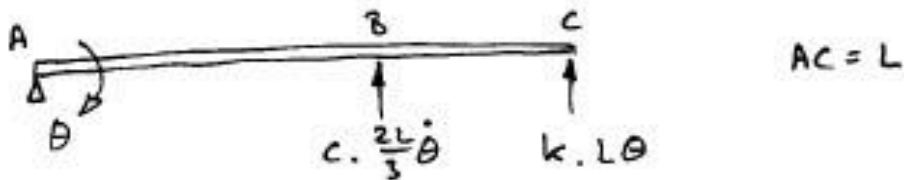
$$\text{If } \delta \ll 1, T_n \approx \frac{2\pi}{\omega_n} = 0.281 \text{ s}$$

$$\text{From ① } \log_e 4 = \delta \omega_n \cdot 3T$$

$$\therefore \delta = 0.0735 \text{ (which is } \ll 1)$$

$$\text{Damping coefficient, } c = \delta \cdot c_{\text{CRIT}} = \delta \cdot 2\sqrt{km}$$

5.



$$\text{A.C.) } -\left(c \frac{2L}{3} \dot{\theta}\right) \cdot \frac{2L}{3} - (kL\theta) \cdot L = I_A \ddot{\theta}$$

$$\text{or } I_A \ddot{\theta} + c \frac{4L^2}{9} \dot{\theta} + kL^2 \theta = 0$$

$$\text{or } 10 \ddot{\theta} + 200 \dot{\theta} + 56,250 \theta = 0$$

$$\text{c.f. } M \ddot{z} + C \dot{z} + Kz = 0$$

$$\omega_n = \sqrt{\frac{K}{M}} = 75 \text{ rad/s} \quad \delta = \frac{C}{2\sqrt{KM}} = 0.133$$

Damped natural frequency,  $\Omega_d = \omega_n \sqrt{1 - \delta^2} = 74.3 \text{ rad/s}$   
 General solution is  $\theta(t) = e^{-\delta\omega_n t} (A \cos \Omega_d t + B \sin \Omega_d t)$

Initial conditions:  $\theta = \theta_0$ ,  $\dot{\theta} = 0$  at  $t = 0$

Hence  $A = \theta_0$  and  $B = \frac{\delta\omega_n}{\Omega_d} \theta_0$

Here,  $\theta_0 = \frac{0.01}{1.5} = 6.67 \text{ mrad}$

Period of damped vibration =  $\frac{2\pi}{\Omega_d} = 0.0845 \text{ s}$

Maximum upward displacement occurs  $\frac{1}{2}$  cycle after the start — that is, after  $0.423 \text{ s}$

Hence,  $\theta = -4.369 \text{ m rad}$  and tip displacement =  $6.55 \text{ mm}$

6. This is an old examination question.

The equation of motion is

$$\left(m + \frac{I}{r^2}\right) \ddot{x} + 2c \dot{x} + \left(2k + \frac{K}{r^2}\right) x = 0$$

where  $m = \text{mass of rack}$

$I = \text{moment of inertia of pinion}$

$c = \text{damping coefficient between rack and ground}$

$k = \text{spring stiffness}$  " " " "

$K = \text{torsional stiffness of pinion shaft}$

$x = \text{rack displacement.}$

This gives  $2.5 \ddot{x} + 100 \dot{x} + 58,000 x = 0$

\* If you have difficulty, go to the next Subject Tutorial

General solution is

$$x(t) = e^{-\gamma \omega_n t} (A \cos \Omega_d t + B \sin \Omega_d t) \quad \textcircled{1}$$

Initial conditions:  $x = 0$ ,  $\dot{x} = 1 \text{ m/s}$  at  $t = 0$

Hence,  $A = 0$  and  $B = 6.62 \text{ mm}$

The maximum displacement occurs approximately  $\frac{1}{4}$  cycle after the start

$$\text{i.e., at } t = \frac{1}{4} \cdot \frac{2\pi}{\Omega_d} = 0.01040 \text{ s}$$

$$\text{This gives } x_{\text{MAX}} = 5.38 \text{ mm}$$

A more accurate value can be found by differentiating  $\textcircled{1}$  to get  $\dot{x}$  and then equating to zero.

$$\text{This gives } t = 0.00953 \text{ s and } x_{\text{MAX}} = 5.43 \text{ mm}$$