



## Additional example 1

Determine the smallest distance  $d$  to the edge of the plate at which the force  $P$  can be applied so that it produces no compressive stresses in the plate at section a-a. The plate has a thickness of 20 mm and  $P$  acts along the centerline of this thickness.

$$\sum F_x = 0, N + (-P) = 0, N = P$$

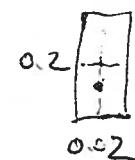
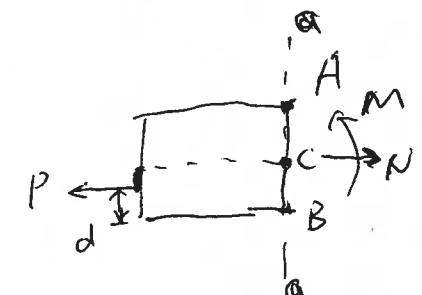
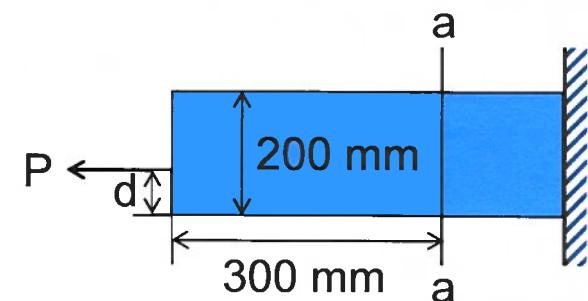
$$\sum M_c = 0, M - P \cdot (0.1 - d) = 0, M = P(0.1 - d)$$

$$top \quad \sigma_A = \frac{N}{A} - \frac{My}{I} = \quad bottom \quad \sigma_B = \frac{N}{A} + \frac{My}{I} \quad I = \frac{l}{12}(0.02) \times 0.2^3 \\ = 13.3333 \times 10^{-6} m^4$$

$$\sigma_A = 0 \Rightarrow 0 = \frac{N}{A} - \frac{My}{I} = \frac{P}{0.2 \times 0.02 \times 200^2} - \frac{P(0.1-d)}{13.3333 \times 10^{-6} m^4}$$

$$\Rightarrow 0 = 250P - 7500P(0.1-d)$$

$$\Rightarrow d = 0.06667 m = 66.7 \text{ mm}$$





## Additional example 2

The horizontal force of  $P = 80 \text{ kN}$  acts at the end of the plate. The plate has a thickness of 10 mm and  $P$  acts along the centerline of this thickness such that  $d = 50 \text{ mm}$ . Plot the distribution of normal stress acting along section a-a.

$$\sum F_x = 0, N + (-P) = 0, N = P = 80 \text{ kN}$$

$$\sum M_c = 0, M - P \cdot (0.1 - 0.05) = 0, M = 80 \text{ kN} \times 0.05 \text{ m} = 4 \text{ kNm}$$

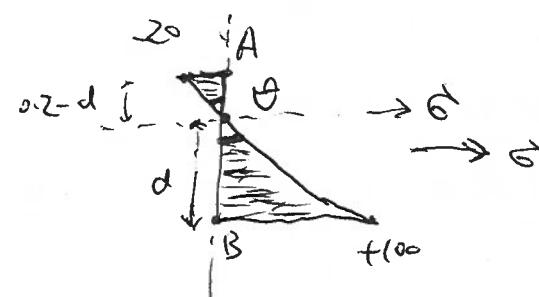
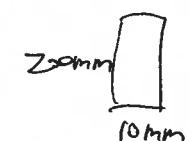
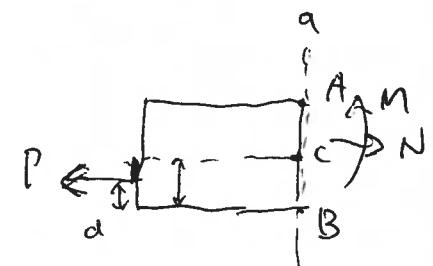
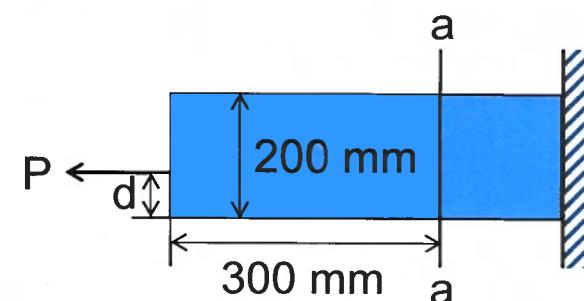
$$A = 0.01 \times 0.2 = 0.002 \text{ m}^2, I = \frac{1}{12} 0.01 \times 0.2^3 = 6.667 \times 10^{-6} \text{ m}^4$$

$$\sigma_A = \frac{N}{A} - \frac{My}{I} = \frac{80 \times 10^3}{0.002} - \frac{4 \times 10^3 \times 0.1}{6.667 \times 10^{-6}} = -20.0 \text{ MPa} \quad \text{compressive}$$

$$\sigma_B = \frac{N}{A} + \frac{My}{I} = \frac{80 \times 10^3}{0.002} + \frac{4 \times 10^3 \times 0.1}{6.667 \times 10^{-6}} = 100 \text{ MPa} \quad \text{tensile}$$

neutral axis

$$\tan \theta = \frac{20}{\frac{1}{2} d - d} = \frac{100}{d}$$





## Additional example 3

At a point on the surface of a machine component the stresses acting on the x face of a stress element are  $\sigma_x = 45 \text{ MPa}$  and  $\tau_{xy} = 15 \text{ MPa}$  (see figure). What is the allowable range of values for the stress  $\sigma_y$  if the maximum shear stress is limited to  $\tau_{\max} = 20 \text{ MPa}$ ?

$$\sigma_x = 45 \text{ MPa}, \quad \tau_{xy} = 15 \text{ MPa}, \quad \sigma_y ? , \quad \tau_{\max} = 20 \text{ MPa}$$

$$R = \tau_{\max} = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\Rightarrow \tau_{\max}^2 = \left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2$$

$$\begin{aligned} \Rightarrow \sigma_y &= \sigma_x \pm 2 \sqrt{\tau_{\max}^2 - \tau_{xy}^2} \\ &= 45 \pm 2 \sqrt{20^2 - 15^2} = 45 \pm 26.5 \end{aligned}$$

$$18.5 \leq \sigma_y \leq 71.5 \text{ MPa}$$

